

Matthew Hood, PE 11500 Benthaven Dr Lakewood, CO 80215	Project	Leshner Schmidt Deck		Job No.
	Subject			Sheet no./rev. 1
		Calc By MOH	Date 10/13/13	

To whom it may concern,

I have reviewed the design of the double 2x8 beam supporting the deck. Based on the following assumptions it is adequate as is.

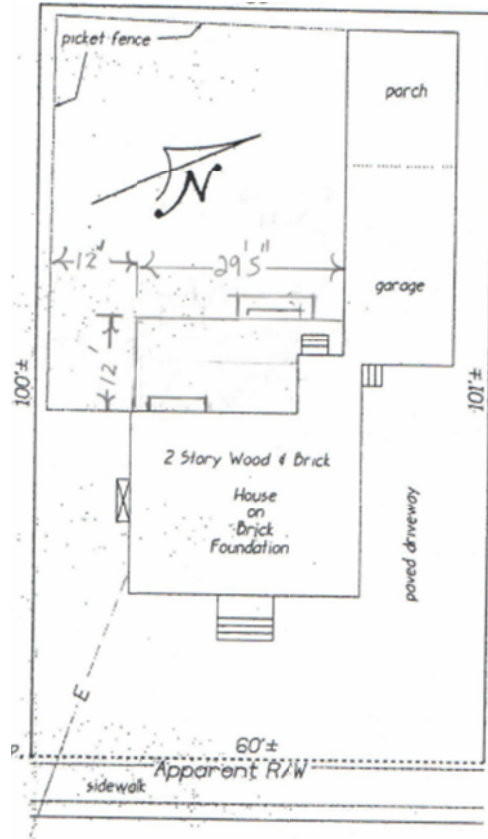
- Dead load is 10 psf
- Maximum snow load on the deck is 50 psf
- The beam is a double 2x8 of yellow pine no. 2
- The maximum clear span is 7'-1"
- The lumber is pressure treated but not incised



See the calculations below for complete justification of this analysis

Sincerely,

Mathew O. Hood, PE



Dead Load;
Snow Load;

DL = 10 psf
SL = 50 psf

Tributary Width;
Span;

Trib = 12 ft / 2 = **6.0** ft
l = 7 ft + 1 in

Load on Beam;
Moment;
Shear;

$\omega = \text{Trib} * (\text{DL} + \text{SL}) = \mathbf{360}$ plf
 $M = \omega * l^2 / 8 = \mathbf{2257.81}$ lb_{ft}
 $V = \omega * l / 2 = \mathbf{1275}$ lbs

Matthew Hood, PE 11500 Benthaven Dr Lakewood, CO 80215	Project	Leshler Schmidt Deck	
	Subject	Sheet no./rev. 2	
		Calc By MOH	Date 10/13/13

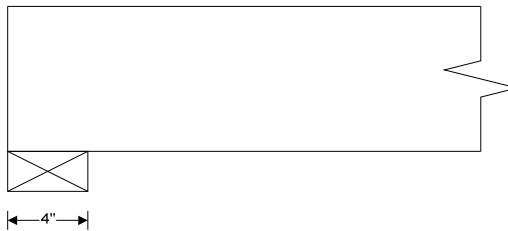
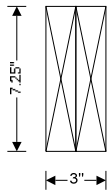
STRUCTURAL WOOD MEMBER DESIGN STRUCTURAL WOOD BEAM DESIGN (NDS 2005)

In accordance with the ASD method

TEDDS calculation version 1.5.09

Analysis results

Design moment in major axis $M_x = 2257 \text{ lb_ft}$
Design shear $F = 1488 \text{ lb}$
Maximum reaction $R = 1488 \text{ lb}$



Sawn lumber section details

Nominal breadth of sections $b_{nom} = 2 \text{ in}$
Dressed breadth of sections $b = 1.5 \text{ in}$
Nominal depth of sections $d_{nom} = 8 \text{ in}$
Dressed depth of sections $d = 7.25 \text{ in}$
Number of sections in member $N = 2$
Overall breadth of member $b_b = N \times b = 3 \text{ in}$

Table 4B - Reference design values for visually graded Southern Pine dimension lumber (2"-4" thick)

Species, grade and size classification Southern Pine, No.2 grade, 8" wide
Bending parallel to grain $F_b = 925 \text{ lb/in}^2$
Tension parallel to grain $F_t = 550 \text{ lb/in}^2$
Compression parallel to grain $F_c = 1350 \text{ lb/in}^2$
Compression perpendicular to grain $F_{c_perp} = 565 \text{ lb/in}^2$
Shear parallel to grain $F_v = 175 \text{ lb/in}^2$
Modulus of elasticity $E = 1400000 \text{ lb/in}^2$
Mean shear modulus $G_{def} = E / 16 = 87500 \text{ lb/in}^2$

Member details

Service condition **Dry**
Length of bearing $L_b = 4 \text{ in}$
Load duration **Two months**

Section properties

Cross sectional area of member $A = N \times b \times d = 21.75 \text{ in}^2$
Section modulus $S_x = N \times b \times d^2 / 6 = 26.28 \text{ in}^3$
 $S_y = d \times (N \times b)^2 / 6 = 10.87 \text{ in}^3$
Second moment of area $I_x = N \times b \times d^3 / 12 = 95.27 \text{ in}^4$
 $I_y = d \times (N \times b)^3 / 12 = 16.31 \text{ in}^4$

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Adjustment factors

Load duration factor - Table 2.3.2	$C_D = 1.15$
Temperature factor - Table 2.3.3	$C_t = 1.00$
Size factor for bending - Table 4B	$C_{Fb} = 1.00$
Size factor for tension - Table 4B	$C_{Ft} = 1.00$
Size factor for compression - Table 4B	$C_{Fc} = 1.00$
Flat use factor - Table 4B	$C_{fu} = 1.15$
Incising factor for modulus of elasticity - Table 4.3.8	$C_{iE} = 1.00$
Incising factor for bending, shear, tension & compression - Table 4.3.8	$C_i = 1.00$
Incising factor for perpendicular compression - Table 4.3.8	$C_{ic_perp} = 1.00$
Repetitive member factor - cl.4.3.9	$C_r = 1.00$
Bearing area factor - cl.3.10.4	$C_b = 1.00$
Depth-to-breadth ratio - Beam is fully restrained	$d_{nom} / (N \times b_{nom}) = 2.00$
Beam stability factor - cl.3.3.3	$C_L = 1.00$

Bearing perpendicular to grain - cl.3.10.2

Design compression perpendicular to grain	$F_{c_perp}' = F_{c_perp} \times C_t \times C_i \times C_b = 565 \text{ lb/in}^2$
Applied compression stress perpendicular to grain	$f_{c_perp} = R / (N \times b \times L_b) = 124 \text{ lb/in}^2$
	$f_{c_perp} / F_{c_perp}' = 0.219$

PASS - Design compressive stress exceeds applied compressive stress at bearing

Strength in bending - cl.3.3.1

Design bending stress	$F_b' = F_b \times C_D \times C_t \times C_L \times C_{Fb} \times C_i \times C_r = 1064 \text{ lb/in}^2$
Actual bending stress	$f_b = M_x / S_x = 1031 \text{ lb/in}^2$
	$f_b / F_b' = 0.969$

PASS - Design bending stress exceeds actual bending stress

Strength in shear parallel to grain - cl.3.4.1

Design shear stress	$F_v' = F_v \times C_D \times C_t \times C_i = 201 \text{ lb/in}^2$
Actual shear stress - eq.3.4-2	$f_v = 3 \times F / (2 \times A) = 103 \text{ lb/in}^2$
	$f_v / F_v' = 0.510$

PASS - Design shear stress exceeds actual shear stress